

1 **Development of an adaptive CTM-RPIM method for modelling**
2 **large deformation problems in geotechnical engineering**

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22 **Abstract**

23 In this paper, a meshfree method namely adaptive CTM-RPIM is developed for modelling
24 geotechnical problems with large deformations. The developed adaptive CTM-RPIM is a
25 combination of the Cartesian transformation method (CTM), the radial point interpolation method
26 (RPIM) and the alpha shape method. To reduce the requirement of meshes, the CTM is adopted to
27 transform domain integrals into line integrals and RPIM is applied to construct interpolation
28 functions. The alpha shape method, which is capable of capturing the severe boundary evolution due
29 to large deformations, is then introduced into the CTM-RPIM to form the adaptive CTM-RPIM. The
30 accuracy of CTM-RPIM is first verified by considering a cantilever beam under small deformation,
31 where the influence of key parameters on simulation results is explored. Afterwards, the ability of
32 the adaptive CTM-RPIM for handling large deformation problems is demonstrated by simulating
33 cantilever beams with large deformations for which analytical solutions are available. In addition, a
34 slope failure problem and a footing bearing capability problem are modelled to illustrate the ability
35 of evaluating the stability of geotechnical structures. Finally, a 2-D soil collapse experiment using
36 small aluminum bars is simulated to test its capability of simulating geotechnical large deformation
37 problems. These benchmark examples show that the adaptive CTM-RPIM is a numerical method of
38 broad application prospect.

39

40 **Keywords:** Cartesian transformation method; Radial point interpolation method; Alpha shape method;
41 Large deformation; Geotechnical engineering.

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51 **1. Introduction**

52 The occurrence of many natural disasters, such as landslides, earthquakes, debris flows and so on,
53 are often accompanied by large and severe deformations [1-3]. Many numerical methods have been
54 developed to solve these problems, but some of them may encounter some problems, such as mesh
55 entanglement and inconsistency between integral domain and problem domain, etc.

56 When dealing with large deformation problems, to some mesh based methods such as the finite
57 element method (FEM) [4, 5] and the finite difference method (FDM) [6, 7], remesh, a time
58 consuming operation, is necessary to avoid mesh distortion. To overcome the reliability of mesh,
59 various meshfree methods have been developed. However, background mesh is used in some
60 classical meshfree method to facilitate the integration of stiffness matrix and force vector etc. Take
61 the material point method (MPM) [8, 9] and the element free Galerkin method (EFGM) [10-12] for
62 example, the use of background mesh will result in the inconsistency between the integral domain
63 and the problem domain, which may cause errors when the model is rough. In addition, this may
64 bring up a problem in MPM that may increase the computational cost, that is, at each step, you need
65 to judge which background cell the integral point is located in. Some meshfree methods without
66 background mesh may also have minor disadvantages, such as the meshfree local Petrov-Galerkin
67 method (MLPG) [13]. In MLPG, the weak form of governing equation is established by local
68 Petrov-Galerkin method, which may lead to the asymmetry of stiffness matrix and increase the
69 computational time. Summing up the above, a meshfree method without background mesh, using
70 Galerkin method to construct its weak form and having coincident problem domain and integral
71 domain can be developed to avoid these disadvantages.

72 The Cartesian transformation method (CTM) [14, 15], originating from the boundary element
73 method (BEM) [16], is an integral method that can transform domain integration into a boundary
74 integration and a 1-D integration. This means that mesh or background mesh will not be needed
75 when computing the integration of stiffness matrix and force vector, and the integral domain will
76 coincides with the problem domain, if CTM is used. The radial point interpolation method (RPIM)
77 [17-19] is of high precision for scattered data interpolation, which can get rid of the restriction of
78 mesh when constructing the shape functions. Moreover, RPIM has a simple theory basis, which is
79 easy to implement with programming and extend to 3-D space. It is conceivable that the combination

80 of CTM and RPIM in the governing equation based on Galerkin method would result in a truly
81 meshfree method with symmetrical stiffness matrix and coincident problem domain and integral
82 domain.

83 Another problem in dealing with geotechnical large deformation problems is to track the
84 boundary of problem domain when severe deformations occur. The boundary tracking accuracy has
85 great influence on the result, and one of the easiest ways to search the boundary (surface) of point
86 cloud is using the alpha shape method. The reliability of alpha shape method [20] has already been
87 proved in particle finite element method (PFEM) [21] and smooth particle finite element method
88 (SPFEM) [22-24], in which alpha shape method is used to identify the boundary with Delaunay
89 triangularization. The adaptive procedure can be constructed by tracking the boundary as the growth
90 of deformation.

91 In this paper, to solve the geotechnical large deformation problems, an adaptive CTM-RPIM is
92 formulated, which combines CTM, RPIM and the alpha shape method. The motivation of developing
93 this new method, is to try to incorporate the merits of each method. RPIM basically is a meshfree
94 method, which can model the large deformations, but the key problem is to find an appropriate support
95 domain for each integration point to ensure the numerical stability. Mesh-based method has its
96 advantages in dealing with the numerical integrations, however, it may encounter the mesh
97 entanglement, or the strong rely on the background mesh, resulting in problems of the accurate capture
98 of the material domain as shown in MPM. CTM is a way of transferring the domain integrations into
99 the line integrals, which greatly reduce the requirements of searching for a support domain of each
100 integration point. Thereby, by combining the two methods, it still inherits the advantages of the
101 meshfree method for its capabilities of modelling the large deformations, more importantly, by
102 utilizing the CTM integration algorithm, the numerical stabilities can be improved with a relatively
103 convenient way, i.e. transferring the domain integration into the line integrals. An external algorithm,
104 i.e. the alpha shape method, is incorporated, which allows the accurate track of the boundary for the
105 dynamic process of large deformation. It is noted that, the boundary identification may be avoided, via
106 the use of CTM integration in regular regions inside the object, and nodal integrations outside this
107 region, as seen in EFGM [1].

108 The whole paper is mainly organized in three parts. Firstly, a brief introduction of the adaptive
109 CTM-RPIM is given first, and the formulation of CTM and RPIM as well the procedure of alpha

110 shape method is then presented. Key parameters of CTM and RPIM, regarding to the computational
111 accuracies, are studied, and the optimal parameters are determined and suggested. Secondly, two
112 large deformation examples of cantilever beams are carried out to verify the advantages of this
113 method for large deformation problems. Finally, three geotechnical examples including slope
114 stability, foundation capability and soil flow are shown, which proves the reliability of this method.

115

116 **2. Formulation and implementation procedure of the adaptive CTM-RPIM**

117 A detailed introduction to the development of the adaptive CTM-RPIM is carried out in this section.
118 The governing equation and the implementation procedure of adaptive CTM-RPIM are provided to
119 help reads to understand how it works. After that, the computational steps for programming are
120 presented, which include both quasi static version and dynamic version. Finally the formulations of
121 CTM, the construction of RPIM shape functions and the procedure of alpha shape method are
122 illustrated in detail respectively for further understanding to the reader.

123

124 **2.1. Implementation procedure of adaptive CTM-RPIM**

125 **2.1.1 The governing equation**

126 The mechanical behavior of soil usually obeys the governing equation of continuum mechanics,
127 which can be derived from the momentum conservation equation.

$$128 \quad \nabla \boldsymbol{\sigma} + \mathbf{b} = \rho \ddot{\mathbf{u}} \quad (1)$$

129 Where ∇ is the partial differential operator; $\boldsymbol{\sigma}$ is the stress vector; \mathbf{b} is the boundary condition;
130 ρ is the density of material and $\ddot{\mathbf{u}}$ is the acceleration.

131 To solve this partial differential equation (PDE), Galerkin method can be used to form its weak
132 form by introducing a test function $\delta \mathbf{u}$ in to Eq. (1).

$$133 \quad \int_{\Omega} \delta \mathbf{u} : \boldsymbol{\sigma} d\Omega - \int_{\Omega} \delta \mathbf{u} \cdot \mathbf{f}^b d\Omega - \int_{\Gamma} \delta \mathbf{u} \cdot \mathbf{f}^{\Gamma} d\Gamma = \int_{\Omega} \delta \mathbf{u} \cdot \rho \ddot{\mathbf{u}} d\Omega \quad (2)$$

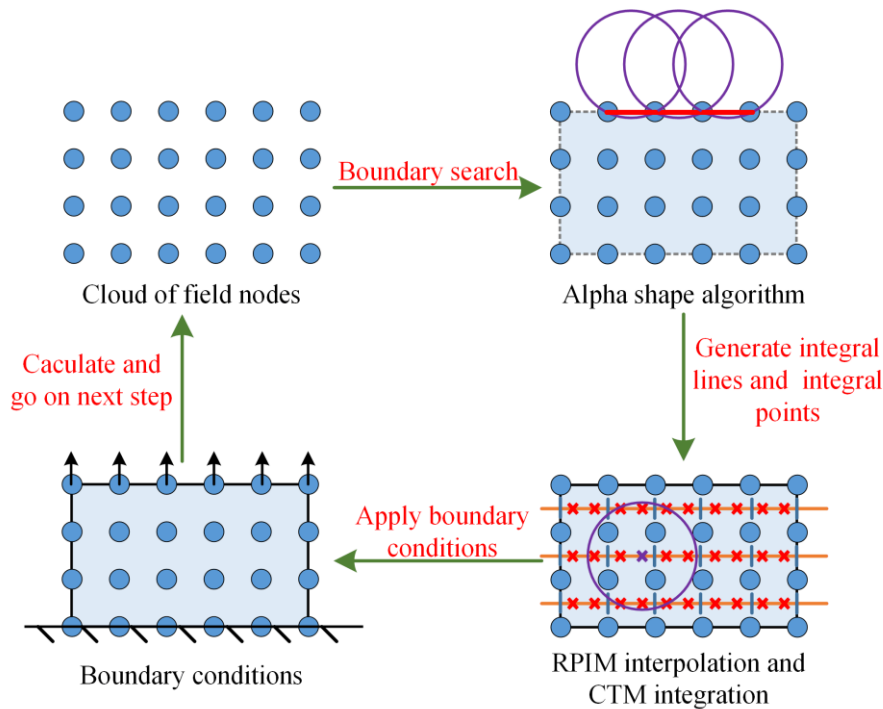
134 Where \mathbf{f}^b is the body force and \mathbf{f}^{Γ} is the surface force. The above equation is a conservation
135 equation of virtual energy. The first term at the left-hand side is caused by internal force, the second
136 and third term is caused by external force and the term at the right-hand side is caused by inertia

137 force.

138 To solve Eq. (2), the discrete form should be constructed just like the way in standard FEM, and
139 the adaptive procedure should be established to deal with large deformation problems. These will be
140 given in the next three section.

141 2.1.2 Adaptive procedure for large deformations

142 How to combine the alpha shape method, CTM and RPIM to construct an adaptive procedure that
143 can be automatically executed during large deformations will be explained in this section. In this
144 procedure, the CTM-RPIM is used to establish the discrete form of momentum conservation
145 equation and the alpha shape method is used to track the boundary when the configuration is updated.
146 A simple schematic diagram is given in Fig.1 to help readers to understand this procedure.



147

148 Fig. 1. The implementation procedure of adaptive CTM-RPIM

149 As shown in Fig.1, the adaptive CTM-RPIM for large deformation analysis can be divided into
150 four basic steps.

151 (1) Search the boundary of the field nodes cloud to determine the scope of problem domain and
152 integral domain.

153 (2) Generate integral lines and integral points, and form the stiffness matrix, mass matrix and
154 force vector.

155 (3) Apply boundary conditions to establish the discretized equation to be solved.

156 (4) Solve the equation formed in step (3), then delete the previous boundary, and the problem
157 domain still returns to the state represented by field nodes.

158 Repeat the above four steps until the error of displacement, force, or energy between two
159 adjacent steps is tolerable.

160 It is clear that remesh operation is not required in the proposed adaptive CTM-RPIM. Only
161 boundary identification is carried out using alpha shape method, so there will be no problems caused
162 by mesh distortion. Moreover, the integral domain is consistent with the problem domain, which is
163 helpful to accuracy. Additionally, in contrast to the EFGM and the traditional RPIM, there is no need
164 to check whether the integral points are in the problem domain. Furthermore, comparing with MPM,
165 there is no need to find which background cell an integral point is in.

166 2.1.3 Computational steps for quasi static problems

167 For quasi static problems, the acceleration can be ignored, so that the right-hand term of Eq. (2) is
168 eliminated. The governing equation can be rewritten as:

$$169 \int_{\Omega} \delta \mathbf{u} : \boldsymbol{\sigma} d\Omega = \int_{\Omega} \delta \mathbf{u} \cdot \mathbf{f}^b d\Omega + \int_{\Gamma} \delta \mathbf{u} \cdot \mathbf{f}^{\Gamma} d\Gamma \quad (3)$$

170 This is a form commonly found in standard FEM, which can be discrete by assuming a relationship
171 between the variables of concerned point and the variables on field nodes using shape functions.
172 Following the similar process of standard FEM, the discretized global equilibrium equation can be
173 derived.

$$174 \mathbf{K} \mathbf{U} = \mathbf{F}_{ext} \quad (4)$$

175 Where \mathbf{K} is the global stiffness matrix, and \mathbf{F}_{ext} is the global external force vector. Using CTM
176 integration the detailed expression of each term can be obtained.

$$177 \mathbf{K} = \sum_{i=1}^m \mathbf{B}_i^T \mathbf{D}_i \mathbf{B}_i w_i^x w_i^y J_i^x J_i^y \quad (5)$$

$$178 \mathbf{F}_{ext} = \sum_{i=1}^m N_i \mathbf{f}_i^b w_i^x w_i^y J_i^x J_i^y + \sum_{j=1}^{mb} N_j \mathbf{f}_j^{\Gamma} w_j J_j \quad (6)$$

179 Where m is the number of integral points, \mathbf{B}_i is the matrix of partial derivative of shape functions
180 at i^{th} integral point, N_i is RPIM shape function matrix at i^{th} integral point, w_i^x , w_i^y , J_i^x , J_i^y are

181 the weight and Jacobi determinant in x and y directions of i^{th} integral point. The second term in \mathbf{F}_{ext}
182 has a similar meaning, except that it is in the form of a boundary integration, which can be solved
183 using Gaussian integral method.

184 Moreover the expression of internal force, which is another expression of \mathbf{KU} in Eq. (4), can also
185 be derived from the left-hand term of Eq. (3).

$$186 \quad \mathbf{F}_{int} = \sum_{i=1}^m \mathbf{B}_i \boldsymbol{\sigma}_i w_i^x w_i^y J_i^x J_i^y \quad (7)$$

187 To solve Eq. (4), Newton-Raphson iteration is often used. And the quasi static version of
188 adaptive CTM-RPIM is given here for material nonlinearity example with small deformation or
189 weightless material with large deformation. The adaptive procedure is as follow

- 190 (1) Discrete the problem domain using a series of field nodes.
- 191 (2) Loop over the incremental step (n^{th} incremental step).
- 192 (3) Search the boundary of problem domain based on the cloud of field nodes using the alpha shape
193 method.
- 194 (4) Generate the integral lines and integral points and form the total external force vector \mathbf{F}_{ext}^n for
195 the present incremental step.
- 196 (5) Loop over Newton-Raphson iteration (l^{th} iteration step).
- 197 (6) Form the global stiffness matrix \mathbf{K}_l^n and the global internal force vector $\mathbf{F}_{int(l)}^n$.
- 198 (7) Solve the governing equation $\mathbf{K}_l^n \mathbf{U}_l^n = \mathbf{F}_{ext}^n - \mathbf{F}_{int(l)}^n$.
- 199 (8) Check convergence.
200 If it is converged, go to step 10.
- 201 (9) End looping the Newton-Raphson iteration.
- 202 (10) Check the incremental step limit.
203 If the incremental step is more than the limit, update the configuration and go to step (12).
204 If the incremental step is less than the limit, update the configuration and go to step (2).
- 205 (11) End looping the incremental step.
- 206 (12) Post processing.

207 2.1.4 Computational steps for dynamic problems

208 For dynamic problems, if weight is taken into account, the acceleration can't be ignored. The left and
 209 right hands of Eq. (2) can be swapped to make it more suitable for constructing the format of time
 210 integral.

$$211 \quad \int_{\Omega} \delta \mathbf{u} \cdot \rho \ddot{\mathbf{u}} d\Omega = \int_{\Omega} \delta \mathbf{u} : \boldsymbol{\sigma} d\Omega - \int_{\Omega} \delta \mathbf{u} \cdot \mathbf{f}^b d\Omega - \int_{\Gamma} \delta \mathbf{u} \cdot \mathbf{f}^{\Gamma} d\Gamma \quad (8)$$

212 In the similar way of standard FEM, the discrete momentum conservation can be derived from Eq.
 213 (8).

$$214 \quad \mathbf{M}\ddot{\mathbf{U}} = \mathbf{F}_{ext} - \mathbf{F}_{int} \quad (9)$$

215 Where \mathbf{M} is the mass matrix, and $\ddot{\mathbf{U}}$ is the acceleration vector. The lumped matrix is used here to
 216 improve the computational efficiency.

217 Sometimes, damping force can be introduced into Eq. (9) for the convergence rate and stability of the
 218 program.

$$219 \quad \mathbf{M}\ddot{\mathbf{U}} = \mathbf{F}_{ext} - \mathbf{F}_{int} + \mathbf{F}_{damp} \quad (10)$$

220 Where \mathbf{F}_{damp} is the damping force vector, which can be expressed as follow, if the local damping is
 221 used.

$$222 \quad \mathbf{F}_{damp} = -f |\mathbf{F}_{ext} - \mathbf{F}_{int}| \text{sign}(\dot{\mathbf{U}}) \quad (11)$$

223 Where f is the damping factor and $\dot{\mathbf{U}}$ is the velocity vector, which can be determined by central
 224 difference method.

225 The detailed procedures with explicit time integral are given as follow for dynamic problems
 226 with both geometric nonlinearity and material nonlinearity.

227 (1) Discrete the problem domain using a series of field nodes.

228 (2) Start time step n .

229 (3) Tracking the boundary using alpha shape method.

230 (4) Generate the integral lines and integral points

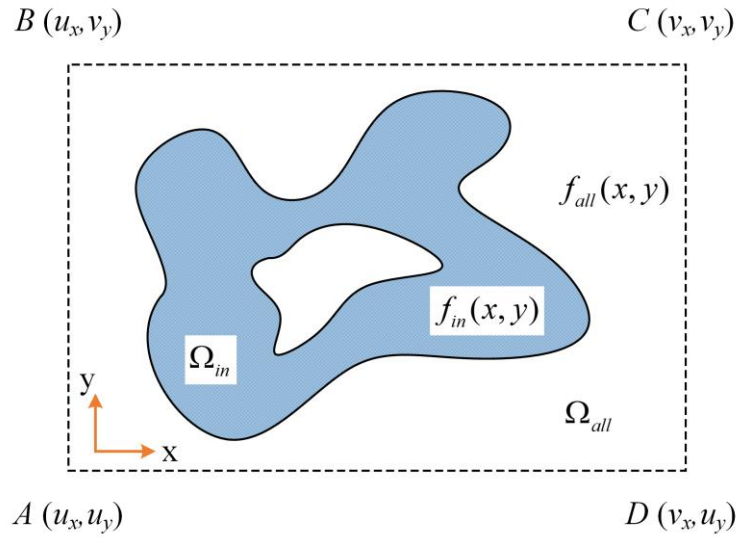
231 A. compute the total force vector at time t^n : $\mathbf{F}^n = \mathbf{F}_{ext}^n - \mathbf{F}_{int}^n + \mathbf{F}_{damp}^n$.

232 B. compute the mass matrix at time t^n : \mathbf{M}^n

233 (5) Calculate the acceleration vector at time t^n : $\ddot{\mathbf{U}}^n = (\mathbf{M}^n)^{-1} \mathbf{F}^n$

- 234 (6) Calculate the velocity vector at time $t^{n+1/2}$: $\dot{\mathbf{U}}^{n+1/2} = \dot{\mathbf{U}}^{n-1/2} + \Delta t^n \ddot{\mathbf{U}}^n$, where $\Delta t^n = t^{n+1/2} - t^{n-1/2}$.
- 235 (7) Calculate the displacement vector at time t^{n+1} : $\mathbf{U}^{n+1} = \mathbf{U}^n + \Delta t^{n+1/2} \dot{\mathbf{U}}^{n+1/2}$, where
- 236 $\Delta t^{n+1/2} = t^{n+1} - t^n$.
- 237 (8) Calculate the internal force vector at time t^{n+1} : \mathbf{F}_{int}^{n+1}
- 238 (9) Update the configuration use the displacement vector \mathbf{U}^{n+1} .
- 239 (10) Check converge or check limit.
- 240 If converged or reached the time step limit, end the calculation, otherwise go to the step (2) and
- 241 start the $n + 1$ time step
- 242

243 **2.2. Formulation of CTM**



244

245 Fig. 2. A domain integration of the function $f_{in}(x, y)$

246 CTM is a special integration method originally applied in BEM [16]. Here, it is introduced into the

247 meshfree method to form the stiffness matrix, mass matrix and force vector. Consider an integration

248 I_{in} of a function $f_{in}(x, y)$ over a domain Ω_{in}

249
$$I_{in} = \int_{\Omega_{in}} f_{in}(x, y) dx dy \quad (12)$$

250 As shown in Fig. 2, the domain is so complex that the integration cannot carried out directly. To

251 solve Eq. (12), a rectangular auxiliary domain Ω_{all} is constructed, which contains fully the domain

252 over which the integration must take place. The function in the rectangular auxiliary domain,
 253 $f_{all}(x, y)$, can then be expressed as

$$254 \quad f_{all}(x, y) = \begin{cases} f_{in}(x, y) & (\text{in } \Omega_{in}) \\ 0 & (\text{out of } \Omega_{in}) \end{cases} \quad (13)$$

255 and the integration I_{in} is rewritten as

$$256 \quad I_{in} = \int_{\Omega_{all}} f_{all}(x, y) dx dy \quad (14)$$

257 Assuming that the function $h_{all}(x, y)$ is the integration of $f_{all}(\xi, y)$

$$258 \quad h_{all}(x, y) = \int_c^x f_{all}(\xi, y) d\xi \quad (15)$$

259 Where ξ is a variable independent of x and y and c is an arbitrary constant. Adopting Green's
 260 theorem, Eq. (14) can be expressed as

$$\begin{aligned} I_{in} &= \int_{\Gamma_{all}} h_{all}(x, y) dy \\ &= \int_{\Gamma_{all}} \left(\int_c^x f_{all}(\xi, y) d\xi \right) dy \\ &= \int_{\Gamma_{AB+BC+CD+DA}} \left(\int_c^x f_{all}(\xi, y) d\xi \right) dy \end{aligned} \quad (16)$$

262 For a rectangular auxiliary domain Ω_{all} , dy is zero on boundary DA and BC . By setting c to be u_x ,
 263 $\int_c^x f_{all}(\xi, y) d\xi$ will be zero on boundary AB implying that the integration in Eq. (16) only needs to
 264 be calculated on boundary CD

$$\begin{aligned} I_{in} &= \int_{\Gamma_{CD}} \left(\int_{u_x}^x f_{all}(\xi, y) d\xi \right) dy \\ &= \int_{\Gamma_{CD}} \left(\int_{u_x}^{v_x} f_{all}(\xi, y) d\xi \right) dy \\ &= \int_{u_y}^{v_y} \left(\int_{u_x}^{v_x} f_{all}(x, y) dx \right) dy \end{aligned} \quad (17)$$

266 The above integration can be divided as following

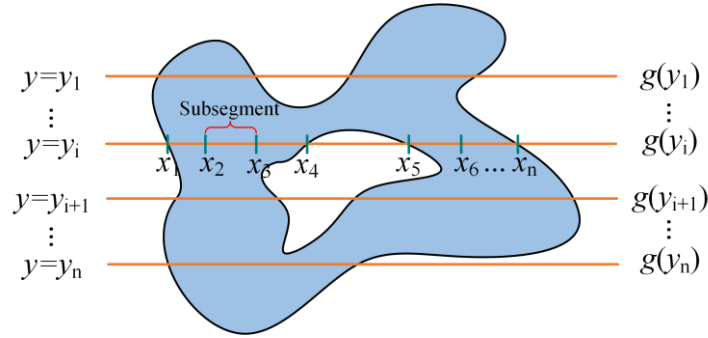
$$267 \quad I_{in} = \int_{u_y}^{v_y} g(y) dy \quad (18)$$

$$268 \quad g(y) = \int_{u_x}^{v_x} f_{all}(x, y) dx \quad (19)$$

269 To calculate these two integrations, a series of integral lines are introduced as shown in Fig. 3.

270 Then the integration I_{in} can be evaluated based on some numerical integration methods, such as

271 Gauss integration, and the value of i^{th} integral line is $g(y_i)$.



272
273 Fig. 3. CTM integral scheme

274 By dividing the i^{th} integral line into $n-1$ segments, $g(y_i)$ can be calculated as

275

$$g(y_i) = \int_{x_1}^{x_n} f_{all}(x, y_i) dx$$

$$= \sum_{j=2}^n \left(\int_{x_{j-1}}^{x_j} f_{all}(x, y_i) dx \right) \quad (20)$$

276 Recalling the relationship between f_{all} and f_{in} , $g(y_i)$ can also be expressed in terms of f_{in} . If

277 f_{in} is used, the integration $\int_{x_4}^{x_5} f_{in}(x, y_i) dx$ vanishes in Fig. 3 and $g(y_i)$ is written as

278

$$g(y_i) = \sum_{j=2}^n \left(\int_{x_{j-1}}^{x_j} f_{in}(x, y_i) dx \right) \quad (j \neq 5) \quad (21)$$

279 Similar to the evaluation of I_{in} , $g(y_i)$ can be calculated numerically. Using Gauss integration

280 scheme, we have

281

$$I_{in} = \sum_{i=1}^{m_y} g(y_i) \cdot w_i^y \cdot J_i^y \quad (22)$$

282 where m_y represents the total number of integral lines, w_i^y and J_i^y are the weight and Jacobi

283 determinant along y direction, respectively, and

284

$$g(y_i) = \sum_{j_x=1}^{m_x} f_{in}(x_{j_x}, y_i) \cdot w_{j_x}^x \cdot J_{j_x}^x \quad (23)$$

285 where m_x is the total number of integral points on each integral line, $w_{j_x}^x$ and $J_{j_x}^x$ are the weight and

286 Jacobi determinant along x direction, respectively.

287 Substituting Eq. (23) into (22) leads to the final form of I_{in}

288

$$I_{in} = \sum_{k=1}^m f_{in}(x_k, y_k) \cdot w_k^x \cdot w_k^y \cdot J_k^x \cdot J_k^y \quad (24)$$

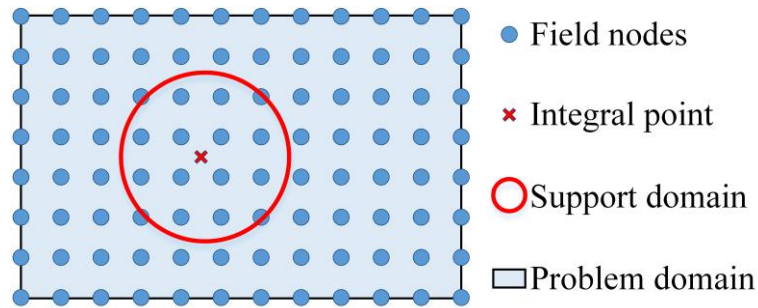
289 which is the sum of the produce of the function at integral points and the corresponding weight and
 290 Jacobi determinant in two directions. Where k is the global number of integral points, and m is the
 291 total number of integral points. As such the CTM trasforms domain integration into line integration.
 292 Consequently, only the intersections of integral lines and boundary are needed in the simulation
 293 leading to a truly meshfree method with coincident problem domain and integral domain. Eq. (24) is
 294 exactly the numerical integration version used in Section 2.1 to caculate the stiffness matrix, mass
 295 matrix and force vector.

296

297 2.3. RPIM based shape functions

298 Generally, the problem domain is represented by a series of field nodes in meshfree methods.
 299 Because of the adoption of the CTM, there is no background mesh and thus shape functions should
 300 be constructed based on the field nodes. In this paper, the RPIM is used for this purpose which is
 301 introduced as below.

302



303

Fig.4. The support domain in RPIM

304 For an arbitrary integral point, a support domain can be formed as shown in Fig. 4. The shape of
 305 the support domain can be circular or rectangular. Supposing that the support domain covers n field
 306 nodes, the field variable, $F(x, y)$, at the concerned point can be approximated by

307

$$F(x, y) = \left[\mathbf{R}^T(r(x, y)) \quad \mathbf{P}^T(x, y) \right] \begin{bmatrix} \mathbf{A} \\ \mathbf{B} \end{bmatrix} \quad (25)$$

308 where $\mathbf{R}^T(r(x, y)) = [r_1(x, y) \quad r_2(x, y) \quad \dots \quad r_n(x, y)]$ is the vector consisting of the radial basis
 309 function (RBF) with $r_i(x, y)$ being the distance between the point at (x, y) and the field node at

310 (x_i, y_i) in two-dimensional cases, $\mathbf{P}^T(x, y) = [1 \ x \ y \ xy \ x^2 \ y^2 \ \dots]$ is the vector of
 311 polynomial basis functions, and \mathbf{A} and \mathbf{B} are row vectors of constants to be determined. Eq. (25)
 312 has to be satisfied at all field nodes. For example, the field variable at j^{th} field node (x_j, y_j) is

$$313 \quad F(x_j, y_j) = [\mathbf{R}^T(r(x_j, y_j)) \ \mathbf{P}^T(x_j, y_j)] \begin{bmatrix} \mathbf{A} \\ \mathbf{B} \end{bmatrix} \quad (26)$$

314 By assembling the above equation for all field nodes, the following matrix form is obtained

$$315 \quad \mathbf{F}_0 = \mathbf{R}_0^T \mathbf{A} + \mathbf{P}_0^T \mathbf{B} \quad (27)$$

316 The detailed form of \mathbf{R}_0 , \mathbf{P}_0 and vector \mathbf{F}_0 are given as follow.

$$317 \quad \mathbf{F}_0 = [F(x_1, y_1) \ F(x_2, y_2) \ \dots \ F(x_n, y_n)]^T \quad (28)$$

$$318 \quad \mathbf{R}_0^T = \begin{bmatrix} r_1(x_1, y_1) & r_2(x_1, y_1) & \dots & r_n(x_1, y_1) \\ r_1(x_2, y_2) & r_2(x_2, y_2) & \dots & r_n(x_2, y_2) \\ \vdots & \vdots & \ddots & \vdots \\ r_1(x_n, y_n) & r_2(x_n, y_n) & \dots & r_n(x_n, y_n) \end{bmatrix} \quad (29)$$

$$319 \quad \mathbf{P}_0 = \begin{bmatrix} 1 & 1 & \dots & 1 \\ x_1 & x_2 & \dots & x_n \\ y_1 & y_2 & \dots & y_n \\ \vdots & \vdots & \vdots & \vdots \end{bmatrix} \quad (30)$$

320 Notice that if there are n field nodes in the support domain and an m -term polynomial basis is used,
 321 there are $m+n$ unknowns in vectors \mathbf{A} and \mathbf{B} . In order to ensure that the solution of Eq. (27) is
 322 unique, the following constraint is assumed.

$$323 \quad \mathbf{P}_0 \mathbf{A} = \mathbf{0} \quad (31)$$

324 With this constraint, Eq. (27) can be enriched as

$$325 \quad \begin{bmatrix} \mathbf{F}_0 \\ \mathbf{0} \end{bmatrix} = \begin{bmatrix} \mathbf{R}_0^T & \mathbf{P}_0^T \\ \mathbf{P}_0 & \mathbf{0} \end{bmatrix} \begin{bmatrix} \mathbf{A} \\ \mathbf{B} \end{bmatrix} \quad (32)$$

326 By solving \mathbf{A} and \mathbf{B} from Eq. (32) and substituting them into Eq. (25), we have the expression of
 327 $F(x, y)$ which is

$$328 \quad F(x, y) = [\mathbf{R}^T(r(x, y)) \ \mathbf{P}^T(x, y)] \begin{bmatrix} \mathbf{R}_0^T & \mathbf{P}_0^T \\ \mathbf{P}_0 & \mathbf{0} \end{bmatrix}^{-1} \begin{bmatrix} \mathbf{F}_0 \\ \mathbf{0} \end{bmatrix} \quad (33)$$

329 The first n terms of $\begin{bmatrix} \mathbf{R}^T(r(x, y)) & \mathbf{P}^T(x, y) \end{bmatrix} \begin{bmatrix} \mathbf{R}_0^T & \mathbf{P}_0^T \\ \mathbf{P}_0 & \mathbf{0} \end{bmatrix}^{-1}$ are the RPIM shape functions for the field
 330 nodes in the support domain, which can be used to construct interpolation functions for the integral
 331 point.

332 There are different types of RBFs such as multi-quadrics (MQ), Gaussian (EXP), thin plate spline
 333 (TPS) and Logrithmic. For simplicity, the TPS RBF

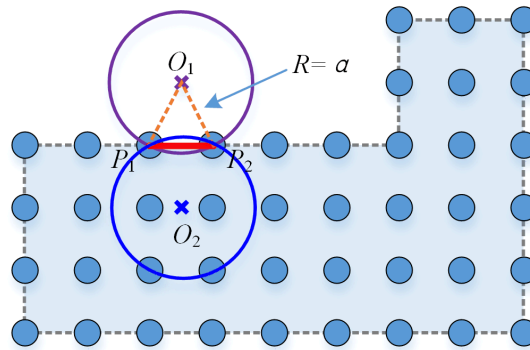
334
$$R(x, y) = r(x, y)^\eta \quad (34)$$

335 is used in this paper where η is the shape parameter.

336

337 **2.4 Alpha shape method for tracking the boundary**

338 As indicated in Section 2.2 and 2.3, a series of field nodes whose boundary is known at the
 339 un-deformed configuration will be generated firstly, if the CTM-RPIM is used. However, when the
 340 object undergoes large deformation the boundary will evolve accordingly. Hence, an efficient
 341 boundary identification method is essential. In this paper, the alpha shape method is adopted for
 342 boundary identification.



343

344 Fig. 5. Schematic diagram of alpha shape method

345 The basic idea of the alpha shape method is to check whether there is an empty circle of radius
 346 α passing through any two field nodes. If such a circle exists, the segment connecting the two field
 347 nodes is a boundary segment. The method can be implemented as follows (see Fig. 5):

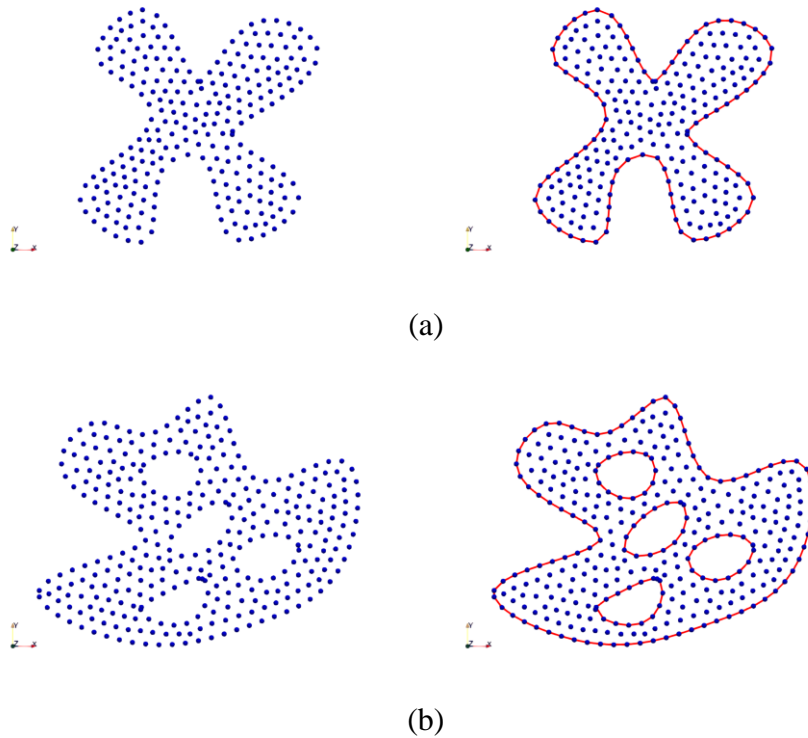
- 348 (1) Calculate the length of line segment P_1P_2 consisting of any two field nodes P_1 and P_2 .
 349 (2) Compare the length of P_1P_2 with the preset circle diameter 2α ;
 350 (3) If P_1P_2 is less than 2α , draw two circles $\odot O_1$ and $\odot O_2$ with radius α passing through P_1

351 and P_2 ;

352 (4) If either circle is empty or with other nodes on the circle, line P_1P_2 is treated as a boundary
353 segment;

354 (5) By repeating steps (1) – (4) for all combinations of field nodes, the boundary of the scattered
355 field node cloud can be obtained.

356 The value of α sometimes has some influence on the tracking accuracy of the boundary, and
357 some experiences can be referenced. In general, for regions of convex polygons, α can be larger,
358 and for regions of concave polygons or with holes, α can be smaller. The boundaries of two sets of
359 field nodes with complex boundaries have been identified using alpha shape method, as shown in Fig.
360 6.



365 Fig. 6 Results of boundary identification (a) Problem domain without hole; (b) Problem domain with
366 holes.

367 As can be seen from Fig. 6, the boundary identified by alpha shape method has reflect the shape
368 of the problem domain well. Predictably, using the alpha shape method, the boundary can be
369 recognized automatically in the simulation regardless of the large change of geometry. This enables
370 the adaption of the method for analyzing large deformation problems.

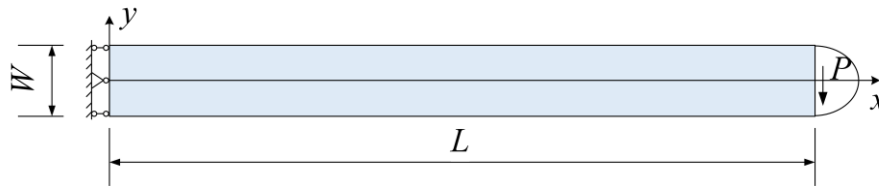
371

372 **3. Numerical examples**

373 In this section, the influence of key parameters on the simulation results of the adaptive CTM-RPIM
 374 is investigated by simulating a cantilever beam subjected to small deformations. Thereafter, the
 375 precision of the adaptive CTM-RPIM for large deformation analysis is verified via modelling
 376 cantilever beams with large deformations. Moreover, the ability of evaluating the stability of
 377 geotechnical structures is tested using a slope and a foundation as benchmark examples. Finally, a
 378 2-D soil collapse experiment is simulated using this method, which further demonstrates the
 379 capability of solving practical geotechnical problems.

380

381 **3.1. Discussions on the selection of key parameters**



382

383 Fig. 7. The cantilever beam with parabolic load on the right end

384 The first example considered is a cantilever beam with a downward load of parabolic distribution on
 385 the right end, as shown in Fig. 7 [25, 26]. The analytical solution of the vertical displacement is

386
$$\Delta_y = \frac{P}{6EI} (3\mu y^2(L-x) + (4+5\mu)\frac{W^2x}{4} + (3L-x)x^2) \quad (35)$$

387 where P is the force on the right end, E is the Young's modulus, μ is the Poisson's ratio, W is
 388 the width of the beam, L is the length of the beam, x and y are the coordinates. The values of these
 389 parameters are illustrated in Table 1.

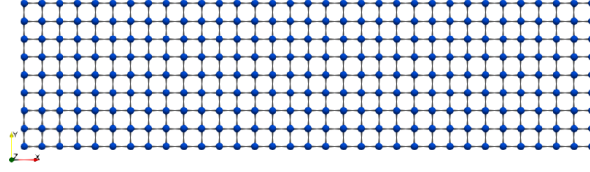
390

Table 1. The parameters for the beam

P	E	μ	W	L
-1000Pa	3×10^7 Pa	0.3	12m	48m

391

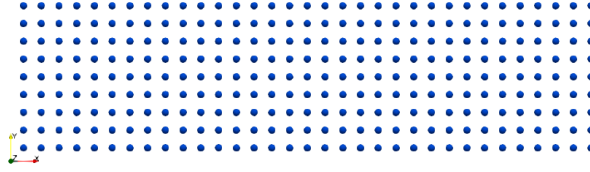
392



393

(a)

394



395

(b)

396 Fig. 8. The model of FEM and CTM-RPIM (33×9 nodes) (a) FEM model; (b) CTM-RPIM model

397 The accuracy of the CTM-RPIM is first verified against the simulation result from FE modelling
 398 with different density levels of nodes. Then the most influential factors are investigated, which are

399 the width between two adjacent integral lines r_y , the length of sub-segment for integration r_x , the

400 radius of support domain r_c and the shape parameter of RPIM η . To measure the distance r_x , r_y

401 and r_c , the mean node spacing d_c is defined

402

$$d_c = \frac{\sqrt{A}}{\sqrt{n-1}} \quad (36)$$

403 where A is the area of problem domain and n is the total number of field nodes.

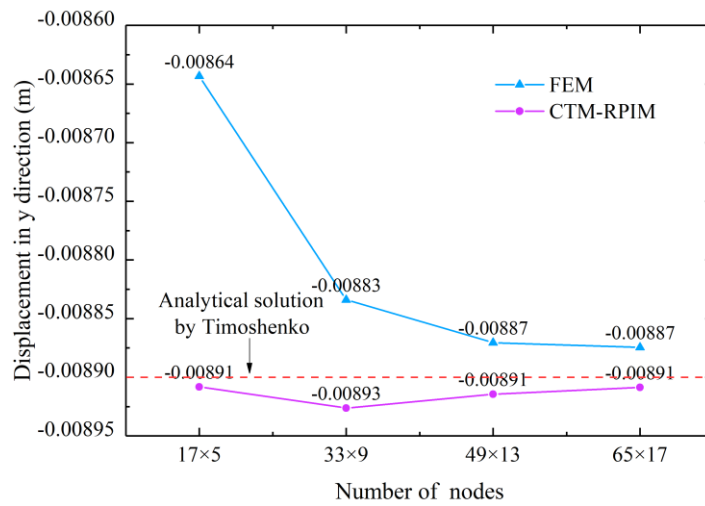
404 The vertical displacements at the right end of the beam from the FEM and CTM-RPIM

405 modelling are shown in Fig. 9. The total number of nodes used in the FEM and CTM-RPIM

406 simulations is the same (Fig. 8). The four-node element is used in FEM (Fig. 8 (a)) while, in the

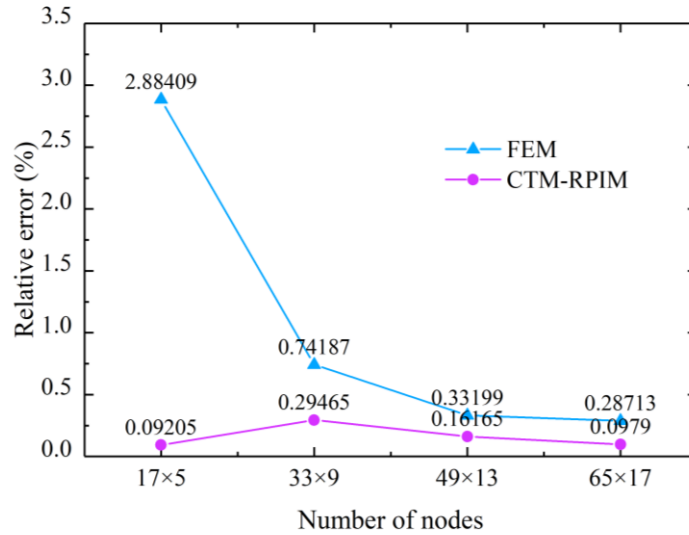
407 CTM-RPIM simulation, we have $\eta = 5.0$, $r_c = 2.0d_c$, and two integration points are used for each

408 integral sub-segment.



409
410

(a)



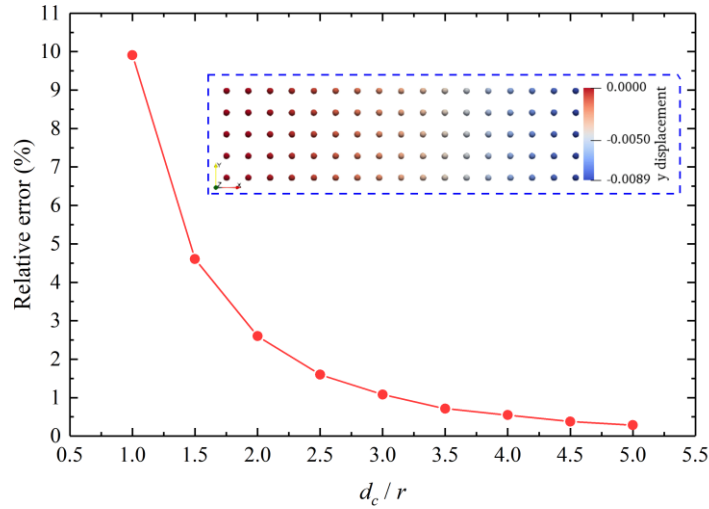
411
412

(b)

413 Fig. 9. Results of vertical displacement at the right end of the cantilever beam (a) The vertical
414 displacement at the right end of the beam; (b) The relative error of the calculated vertical displacement
415 versus the number of used nodes

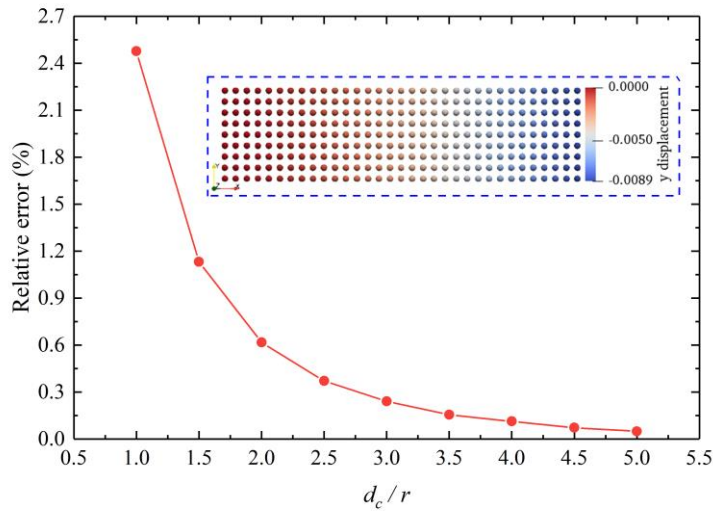
416 As seen in Fig. 9, if the parameters are chosen appropriately, the accuracy of CTM-RPIM can be
417 much higher than that of FEM at all field nodes density level, because that RPIM shape functions are
418 high order and have a larger support domain. Even when a cloud of field nodes with loose density is
419 used, the relative error is still low. The maximum relative error of CTM-RPIM is only about 0.3%.
420 Additionally, the integral points used in CTM-RPIM is less than that of FEM. The ratio between the

421 numbers of integration points used in CTM-RPIM and the FEM is $\frac{a}{2(a-1)}$ where a is the number
 422 of nodes in y direction.



423
424

(a)



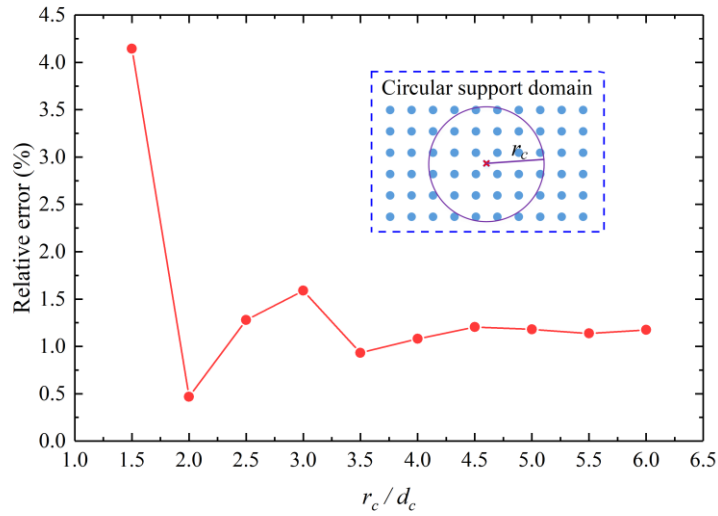
425
426

(b)

427 Fig. 10. The influence of r_x and r_y (a) Results of 17×5 nodes; (b) Results of 33×9 nodes

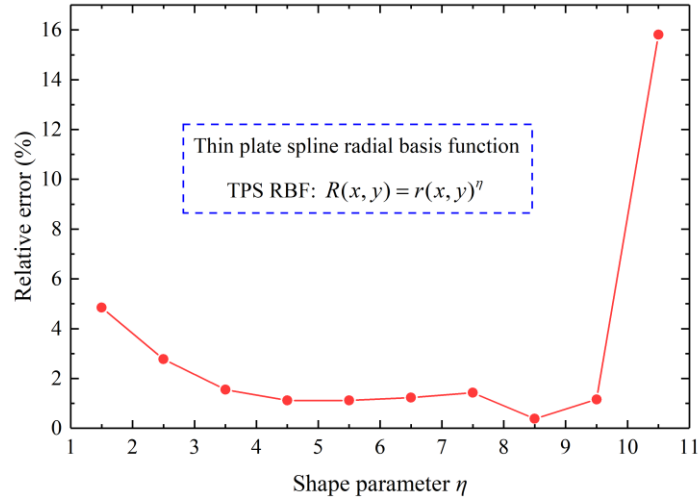
428 The influence of the number of integral points on the CTM-RPIM simulation is studied in this
 429 section. Two sets of field nodes, namely 17×5 and 33×9 , are used in the simulation with $r_x = r_y = r$.
 430 A sufficiently large radius of support domain is selected which is $r_c = 4.0d_c$. The shape parameter is
 431 set to 5.0. Only one integral point is used for each integral sub-segment, which means that the

432 number of used integral points depends on the length r . To study the influence of the integral points,
 433 the ratio d_c/r varies from 1.0 to 5.0 with an interval of 0.5. The simulation results are shown in
 434 Fig. 10. As expected, the accuracy is improved with the increase of d_c/r and the convergence rate
 435 is roughly exponential. When d_c/r is greater than or equal to 3.0, a satisfactory accuracy is
 436 obtained.



437
 438 Fig. 11. The influence of support domain radius r_c

439 The size of support domain is another factor influencing the CTM-RPIM simulation. A larger
 440 support domain implies that more field nodes contribute in the construction of shape functions which
 441 leads to higher accuracy. However, the computational cost also rises because of the increase in the
 442 involved field nodes. Therefore, it is essential to investigate the size of a support domain in relation
 443 to the simulation accuracy and computational cost. As a circular support domain is adopted in this
 444 work, the non-dimensional radius of the support domain r_c/d_c is the indicator of its size which has
 445 been varied from 1.5 to 6.0 with an interval 0.5 in this study. The layout of field nodes is 17×5 , the
 446 shape parameter η is set to 5.0, the distance r_x and r_y are set to $\frac{1}{3}d_c$, and one integral point is
 447 used for each sub-segment. As can be seen from Fig.11, when the radius $r_c \geq 3.5d_c$, the relative error
 448 is less than 1.5% and nearly stable regardless of the increase of r_c/d_c . Thereby, the recommended
 449 range of the radius of a support domain is between $3.5d_c$ and $4.0d_c$.



450

451

Fig. 12. The influence of shape parameter η

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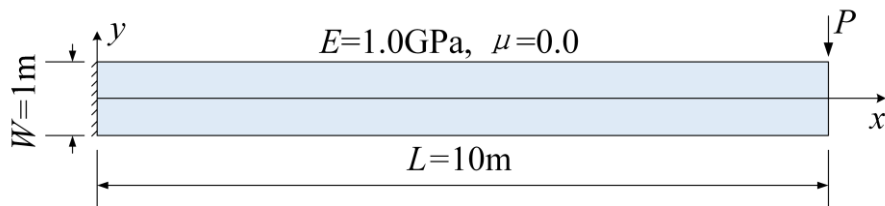
464

465

Finally, the effect of the shape parameter η of TPS RBF on the simulation is explored. To this end, the shape parameters from 1.5 to 10.5 with an interval 1.0 are adopted in the simulation, and other parameters are $r_x = r_y = \frac{1}{3}d_c$ and $r_c = 4.0d_c$. A cloud of filed nodes 17×5 is used to discretize the problem domain and one integral point is assigned to each segment. Fig. 12 shows that the value of η in the range of $[3.5, 9.5]$ leads to higher accuracy, and the relative error is low and stable when η is between 4.5 and 7.5.

3.2. Large deformation analysis of cantilever beam

In this section, the ability of the adaptive CTM-RPIM for dealing with large deformation problems is demonstrated. To this end, two linear-elastic cantilever beams undergo large deformations because of the imposed force (Fig. 13) and moment (Fig. 16) are concerned. The alpha shape method is used to identify the boundary of the problem domain, which makes the CTM-RPIM adaptive regardless of the change of geometry.



465

466

Fig. 13. The cantilever beam with concentrated force on the right end

467

The first case is a cantilever beam subjected to a downward concentrated load P as shown in Fig.

468

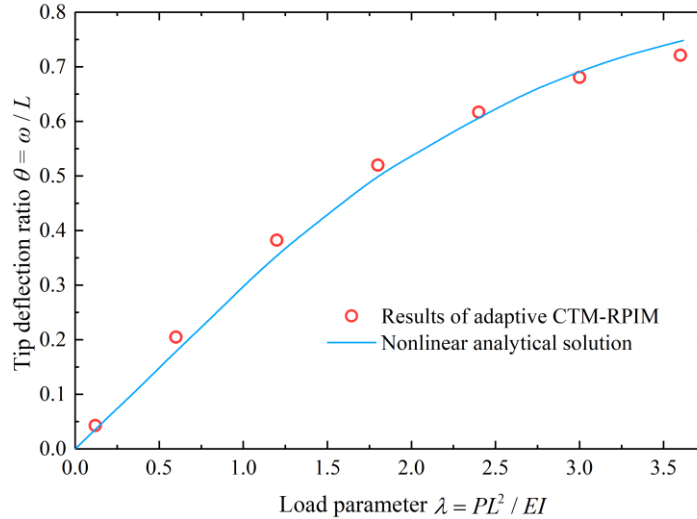
13. The length and the width of the beam are $L=10\text{m}$ and $W=1\text{m}$, respectively. The elastic

469

constants are Young's modulus $E=1.0\text{GPa}$ and Poisson's ratio $\mu=0.0$. A total of 306 field nodes

470

are used in the simulation. The analytical solution of this problem is available in [27].



471

472

Fig. 14. The results at different levels of force P

473

To show the accuracy of the adaptive CTM-RPIM, a series of cases with different concentrated

474

forces at the right-top corner are simulated with results compared to the nonlinear analytical

475

solutions. The applied forces P are at different levels, namely 100kN, 500kN, 1000kN, 1500kN,

476

2000kN, 2500kN and 3000kN respectively. The curves of tip deflection ratio $\theta = \frac{\omega}{L}$, where ω is

477

the deflection at the right end, versus the non-dimensional load parameter $\lambda = \frac{PL^2}{EI}$, where I is

478

the inertia moment of beam section, are shown in Fig. 14. The configuration of the deformed

479

cantilever beam at the concentrated force levels $P=2000\text{kN}$ and $P=3000\text{kN}$ are presented in Fig.

480

15. Clearly, the simulation results from the adaptive CTM-RPIM agree well with the analytical

481

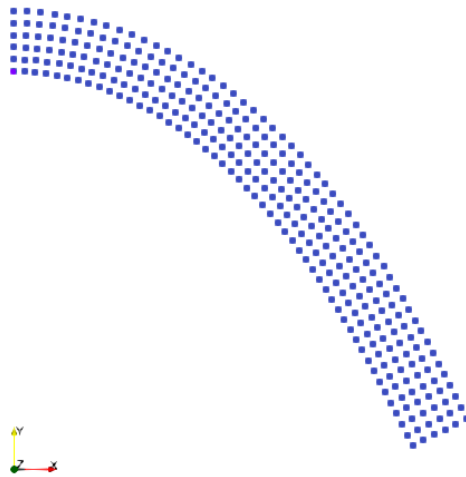
solution from [27], which demonstrates the correctness of the adaptive CTM-RPIM for analyzing

482

large deformation problems.

483

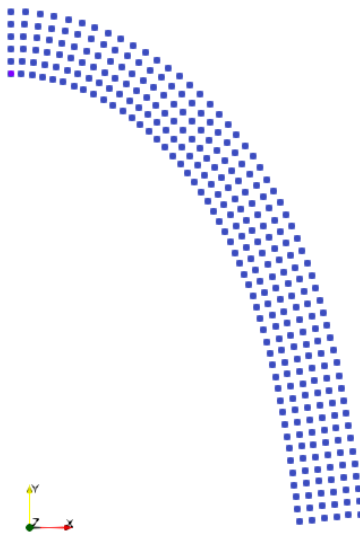
484



(a)

485

486



(b)

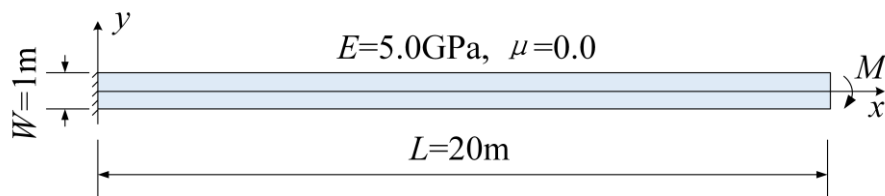
487

Fig. 15. The final configuration of the beam at two different force levels (a) $P = 2000\text{kN}$; (b)

488

$P = 3000\text{kN}$

489



490

491

Fig. 16. The cantilever beam with bending moment on the right end

492

493

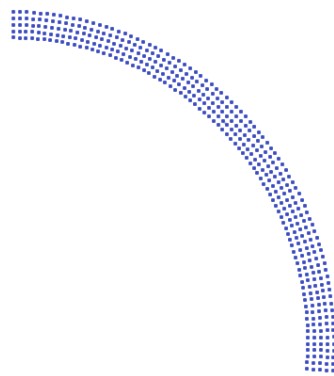
A cantilever beam subjected to a bending moment at the right end is studied in this section. The setup of the problem is shown in Fig. 16. According to the nonlinear analytical solution given by Pai

494 and Palazotto [28], when an appropriate bending moment is applied at the right end, the cantilever
495 beam will bend into a perfect circular ring. The bending moment M to make the beam achieve this
496 state can be determined by

497
$$M = \frac{2\pi EI}{L} \quad (37)$$

498 In the simulation, the moment is transformed into two uniform loads applied to the upper and
499 lower halves of the beam end section according to [29]. A total of 405 field nodes are used to
500 discretize the problem domain and 1000 incremental analysis steps are adopted. The deformation
501 process of the cantilever beam obtained from the adaptive CTM-RPIM is shown in Fig. 17. A near
502 perfect circular ring is obtained when the moment calculated from Eq. (37) is enforced at the end of
503 the beam which verifies the proposed method for large deformation analysis.

504



505

506

(a)



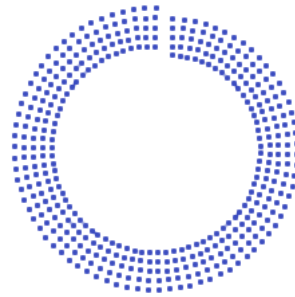
507

508

(b)



(c)



(d)

Fig. 17. The deformation process of the beam (a) 250 steps; (b) 500 steps; (c) 750 steps; (d) 1000 steps

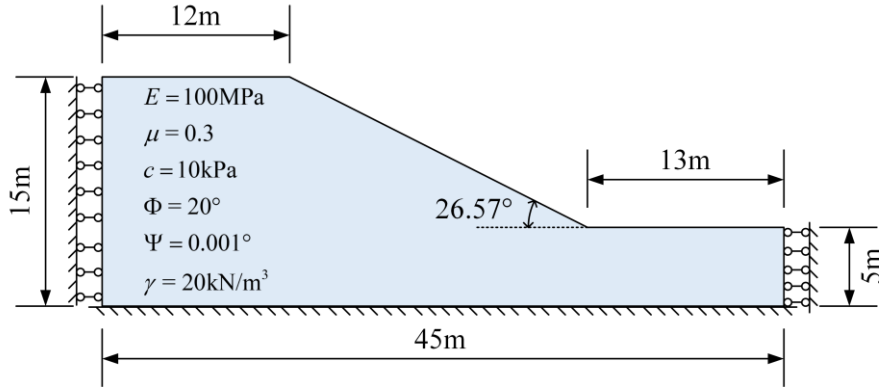
3.3. *Quasi static analysis of stability problem*

Stability analysis is of great importance in geotechnical engineering, which must be carried out to ensure the geotechnical structures won't failure before and after they are finished. To evaluate the capability of adaptive CTM-RPIM in analyze the stability, two classical benchmark examples have been studied in this section. The first example is to evaluate the stability of a slope and the second is to calculate the bearing capacity of a foundation. These two examples fully demonstrate the reliability of this method in the stability analysis of geotechnical engineering.

3.3.1 *Slope stability analysis*

In this section, a homogeneous soil slope is studied with the strength reduction method, and the obtained safety factor is used to verify its reliability. The geometry of the slope is given in Fig. 18, and the material properties are Young's modulus $E = 100\text{MPa}$, Poisson's ratio $\mu = 0.3$, cohesion $c = 10\text{kPa}$, friction angle $\Phi = 20^\circ$, dilatancy angle $\Psi = 0.001^\circ$ and the unit weight $\gamma = 20\text{kN/m}^3$.

528 The bottom of the slope is fully fixed while the lateral boundaries are fixed horizontally. The
 529 elastic-perfectly plastic constitutive model with Mohr-Coulomb yield criterion is used adopted. A
 530 total of 1881 field nodes are used in the CTM-RPIM simulation.



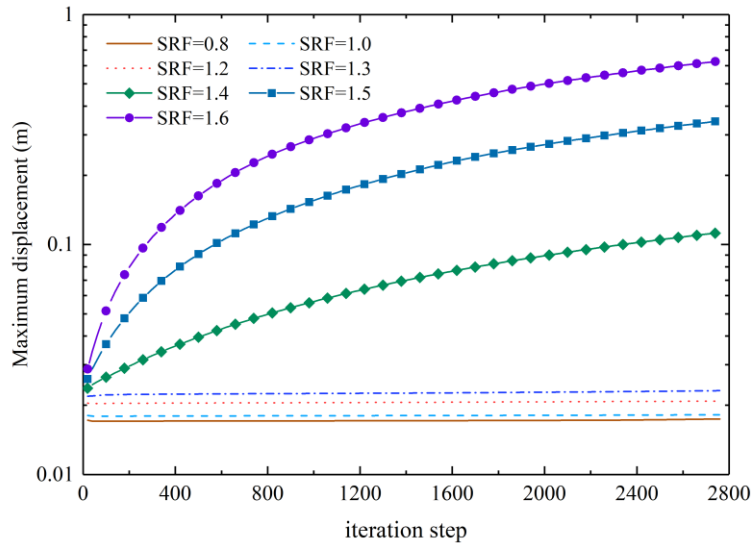
531
 532 Fig. 18. The homogeneous slope

533 The formulation of the strength reduction method [30] can be expressed as

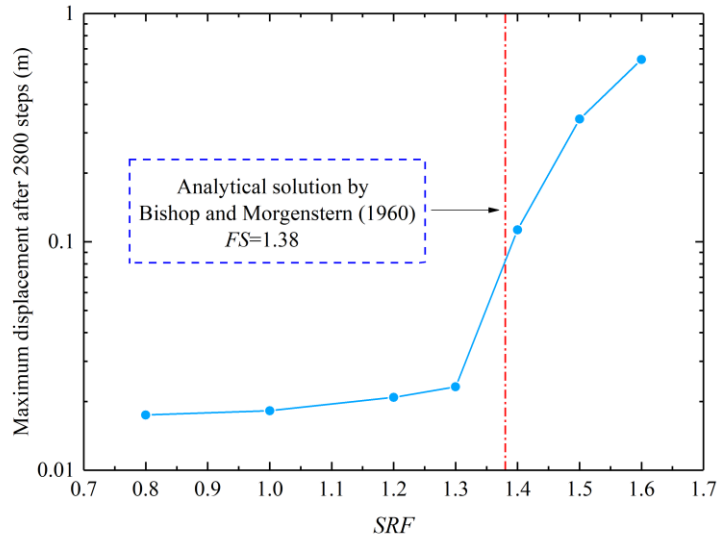
534
$$c' = \frac{c}{SRF} \quad (38)$$

535
$$\Phi' = \arctan\left(\frac{\tan(\Phi)}{SRF}\right) \quad (39)$$

536 where c' and Φ' are the material parameters after strength reduction, and SRF is the reduction
 537 factor.



538
 539 (a)



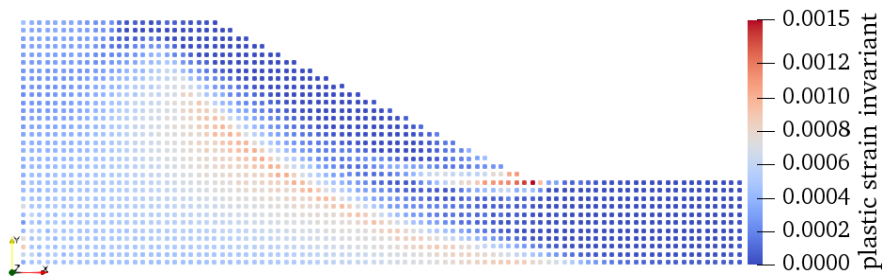
540

541

(b)

542 Fig. 19. The maximum displacement of the slope (a) Maximum displacement for every step; (b) Final
 543 maximum displacement after 2800 steps

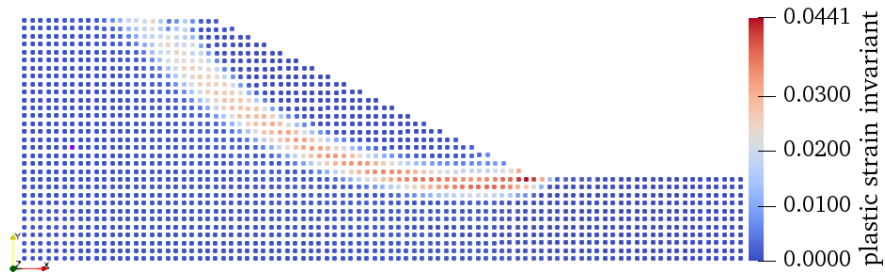
544 The reduction factor SRF ranging from 0.8 to 1.6 is used with simulation results after a large
 545 enough step (2800 steps) illustrated in Fig. 19 (a) and (b). Fig. 19 (a) shows that the deformation is
 546 negligible when $SRF \leq 1.3$. In contrast, the displacement increases continuously and convergence
 547 cannot be achieved for $SRF \geq 1.4$. In other words, when $SRF \leq 1.3$ the slope is stable and when
 548 $SRF \geq 1.4$ the slope is unstable. The same conclusion can also be obtained by showing the curve of
 549 the displacement against SRF (see Fig. 19 (b)). Hence, it is clear that the safety factor FS of the slope
 550 is within (1.3, 1.4), which embraces the analytical solution 1.38 provided by Bishop and
 551 Morgenstern [31].



552

553

(a)



(b)

Fig. 20. The plastic strain invariant of the slope after 2800 steps (a) $SRF = 1.3$; (b) $SRF = 1.4$

Further, to insight why the slope loses stability, the counter of plastic strain invariant of the cases $SRF = 1.3$ and $SRF = 1.4$ are given in Fig. 20. It can be seen that, when the SRF is 1.3 the plastic strain is very small, areas of plasticity are isolated from each other and have not penetrated the whole slope. However, when the SRF is 1.4, the plastic area is connected and form a clear shear band and the soil slides down along the shear band to eventually form a landslide.

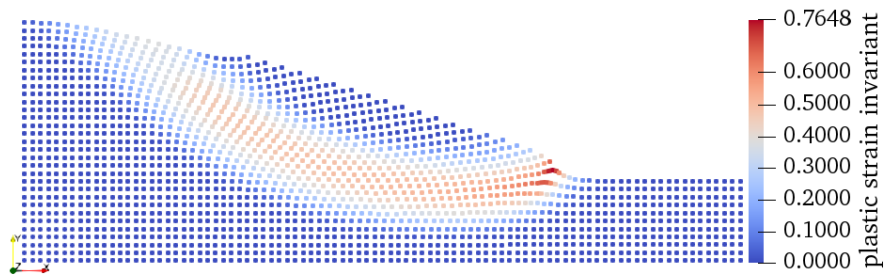


Fig. 21. The result of $SRF = 1.6$ after 10000 steps

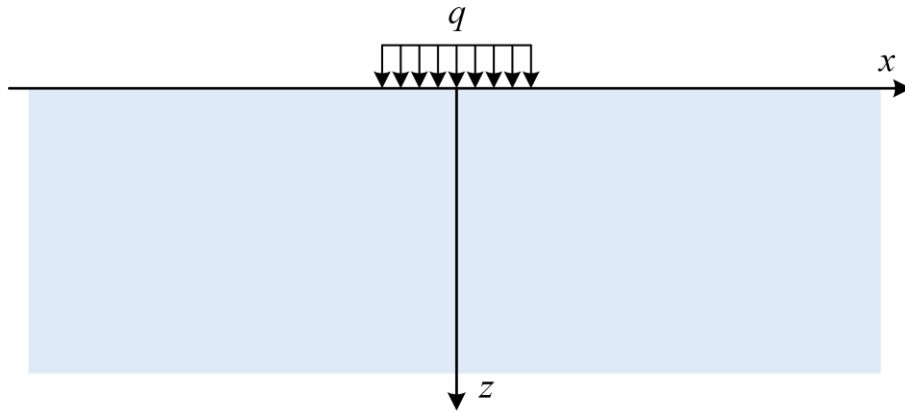
Finally, a large deformation case $SRF=1.6$ after 10000 steps is given in Fig. 21. It preliminarily shows the ability of this method to deal with large deformations, when large deformation occurs, the shear band is still clearly visible without numerical instability

3.3.2 Bearing capacity analysis

A flexible strip footing on weightless soil in semi-infinite space is studied in this section, of which the bearing capacity is analyzed to test its precision, as shown in Fig. 22 (a). The analytical solution of this classical problem was found by Prandtl in 1920 [32]

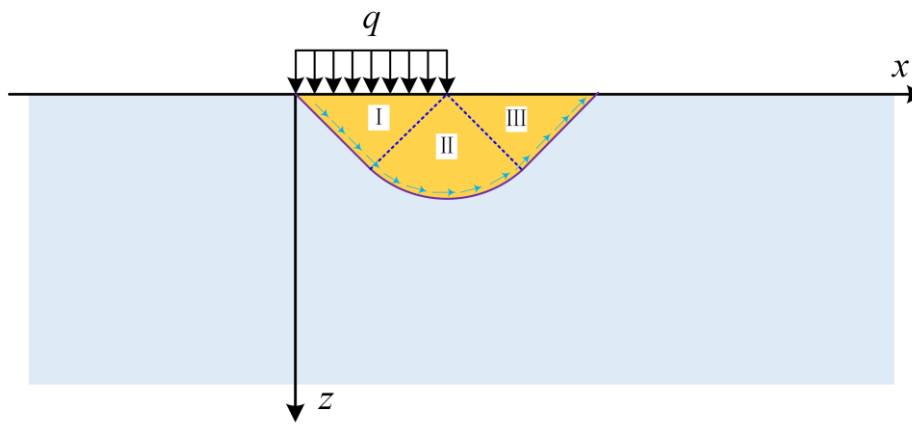
$$q_{failure} = (2 + \pi)c_u \quad (40)$$

where c_u is the undrained shear strength of the soil.



574
575

(a)

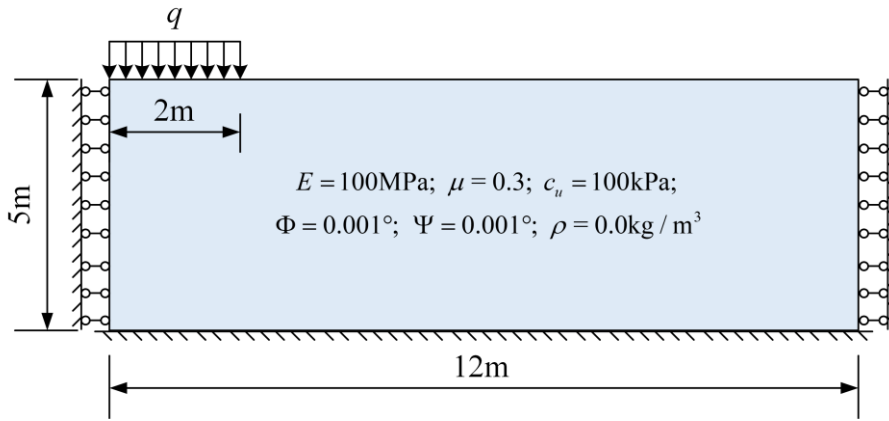


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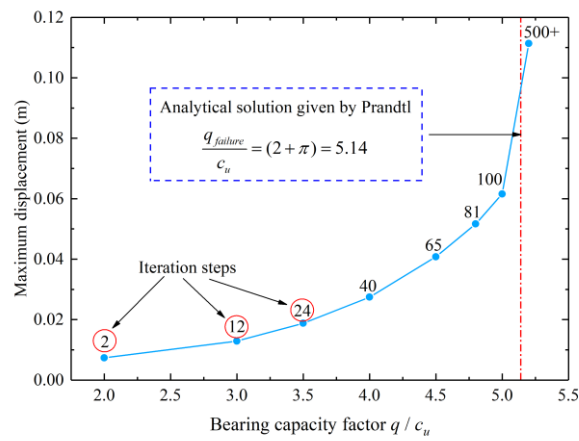
(b)

578 Fig. 22. The flexible strip footing on weightless soil (a) Description of the problem; (b) Prandtl's
579 schematization [32]

580 The simplified model is given in Fig. 23, the scale is $12\text{m} \times 5\text{m}$, the loaded width is 2m , and the
581 boundary conditions are rollers at both side directions and fixed at the bottom. Elastic-perfect plastic
582 constitutive and Mohr-Coulomb yield criterion are used in this example, and the material properties
583 are as follows. Young's modulus $E = 100\text{MPa}$, Poisson's ratio $\mu = 0.3$, the undrained cohesion
584 $c_u = 100\text{kPa}$, friction angle Φ and dilatancy angle Ψ are both set to 0.001° and density ρ is
585 chosen as $0.0\text{kg}/\text{m}^3$. A model with 1029 field nodes is generated to solve this problem, and the
586 result is given in Fig. 24.



587
588 Fig. 23. Symmetry-based simplification for numerical simulation
589



590
591 Fig. 24. Plot of maximum displacement versus bearing stress

592 At different load levels from 200kN to 520kN, the convergence should occur with few iteration
593 steps and small deformation if the strip footing is stable. As can be seen from Fig. 24, when the load
594 is less than or equal to 500kPa, the convergence can be achieved quickly within 100 iteration steps,
595 and the maximum deformation at all load levels is about 6 cm. But when the load reaches 520kPa,
596 the convergence cannot be attained within 500 iterations, and the displacement at 500 steps shown in
597 Fig. 24 is about 11cm, which is nearly two times of that at 500kPa. That means the failure load of
598 the flexible strip footing is between 500kPa and 520kPa, including Prandtl's solution of 514kPa.
599 That is, adaptive CTM-RPIM is reliable in the evaluation of the bearing capacity of the flexible strip
600 footing. If a more accurate result is required, a more refined numerical model can be used.
601

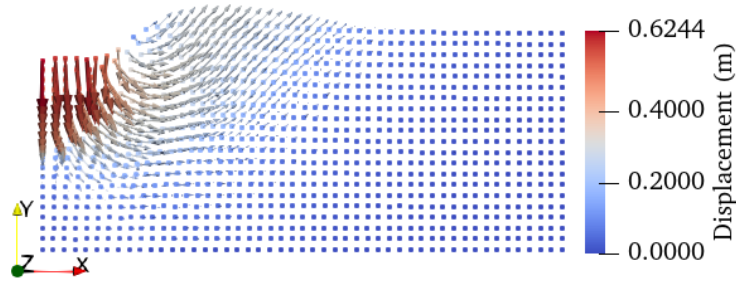


Fig. 25. The displacement at the load level of 540kPa

The deformed configuration at 540kPa is given in Fig. 25, where the deformation reaches over 0.6 m, and the failure mechanism proposed by Prandtl shown in Fig. 22 (b) can be easily seen from the deformation trend displayed by arrows with direction in Fig. 25. Again, it is verified that the adaptive CTM-RPIM has great advantages in stability evaluation and the deformation trend prediction.

3.4 Dynamic analysis of post failure problem

In order to further test the ability of this method for solving geotechnical large deformation problems, the soil collapse process was simulated. It is a 2-D experiment carried out by Ha H. Bui et al. [33], where the soil particles are modeled by many small aluminum bars, as shown in Fig. 26. Constrained by a movable baffle, these bars are initially stacked into a rectangular column. At the beginning of the test, the baffle was quickly removed, and the soil column collapsed rapidly under its own weight. After a long run out distance, it accumulated into an approximate triangle area. The parameters of soil are given as: Young's Module $E = 0.84\text{MPa}$, Poisson's ratio $\mu = 0.3$, cohesion $c = 0\text{MPa}$, friction angle $\Phi = 19.8^\circ$, dilatancy angle $\Psi = 0.001^\circ$ and density $\rho = 2650\text{kg} / \text{m}^3$.

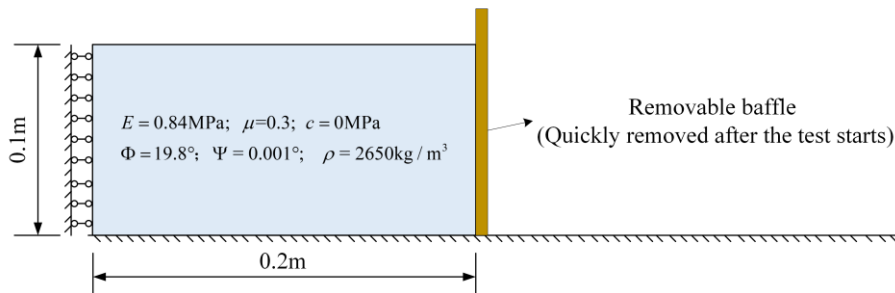
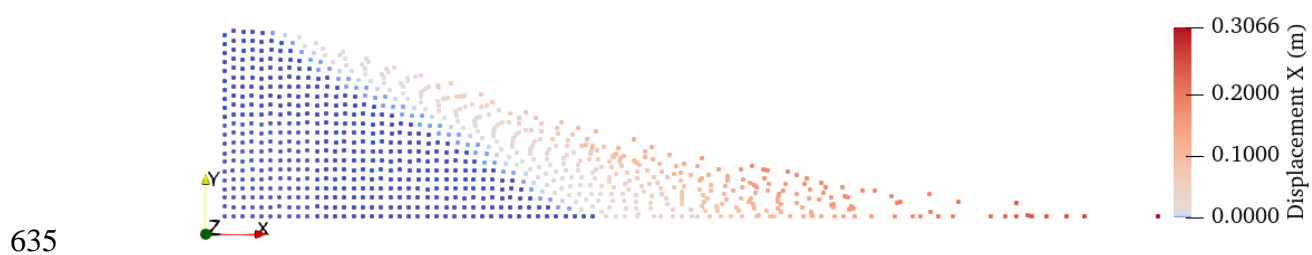
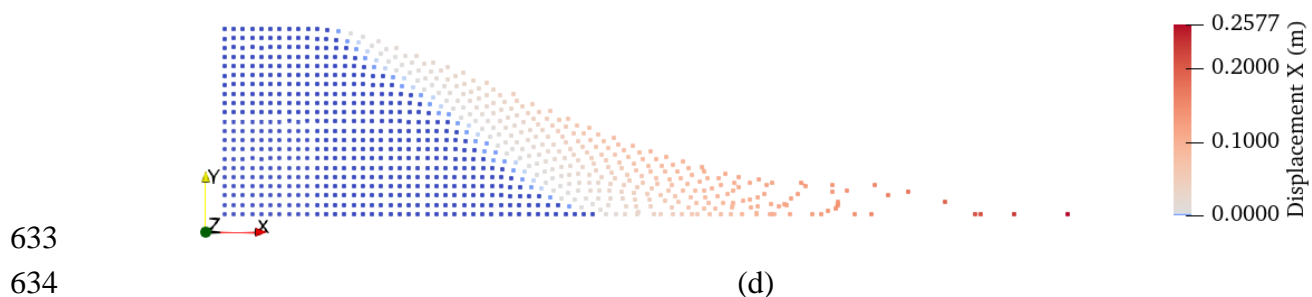
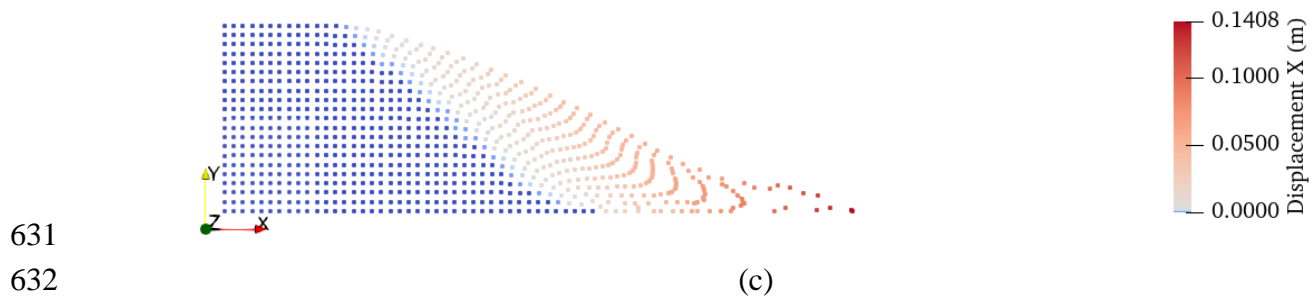
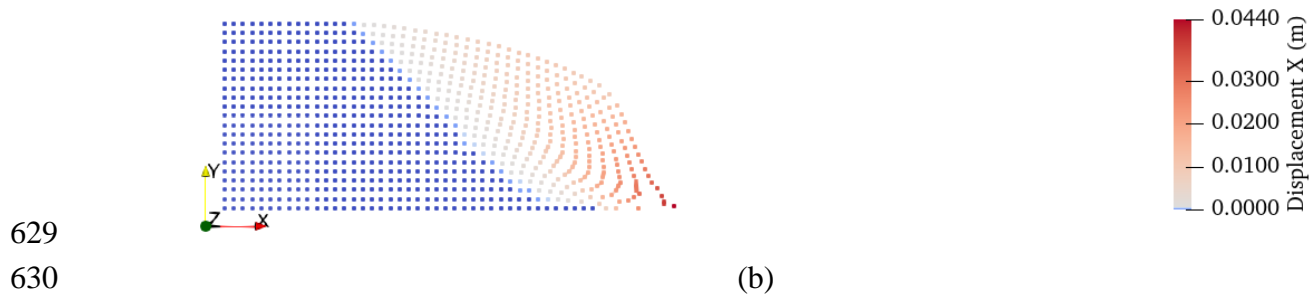
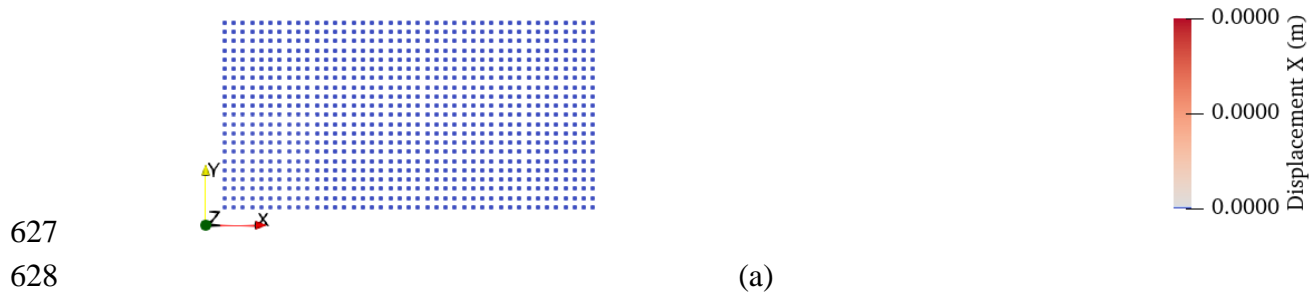


Fig. 26. 2-D experiment of soil collapse [33].

The numerical model was discretized into 1071 field nodes, and elastic-perfect plastic

622 constitutive and Mohr-Coulomb yield criterion are used. The boundary conditions are set as fixed at
623 the bottom boundary and rolling at the left boundary, following the settings given in [33, 34].
624 Because the collapse of soil column is very fast and the deformation is extremely large, the inertia
625 force is not negligible, the dynamic scheme is adopted here. The time step is chosen as $\Delta t = 5.0\mu s$,
626 which is small enough to ensure the stability and accuracy of calculation.

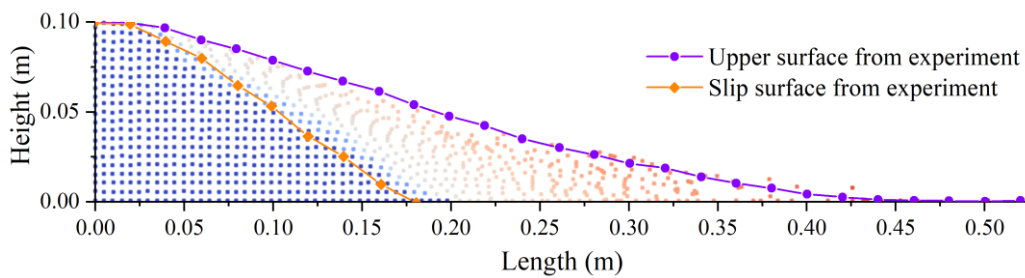


636

(e)

637 Fig. 27. Collapse process of soil column (a) Initial configuration; (b) Configuration at 0.1s; (c)
638 Configuration at 0.2s; (d) Configuration at 0.3s; (e) Final configuration.

639 The collapse process of soil column is given in Fig. 27, in which field nodes are rendered in
640 different colors depending on the displacement to observe the deformation inside the soil column.
641 After the baffle is removed, the soil column immediately begins to collapse, and the soil on the upper
642 right begins to slide, while the soil on the lower left remains static. As time goes on, the static region
643 decreases gradually. At the end of the collapse process, the soil particles stop moving, the upper
644 surface of the accumulation is at an angle slightly less than the friction angle, and a small region
645 remains static in the lower left corner. The interface between the static region and sliding region is
646 what we call sliding surface.



647

648 Fig. 28. Final configuration of the soil collapse example

649 Finally, the final upper surface and the sliding surface are compared with the experimental
650 observations, which is presented in Fig. 28. It can be clearly seen that the numerical result agrees
651 well with the experimental result, which can further demonstrate the ability of adaptive CTM-RPIM
652 for simulating geotechnical large deformation problems.

653

654 4. Conclusions

655 An adaptive CTM-RPIM for solving geotechnical large deformation problems is introduced in this
656 paper. Using CTM and RPIM, the domain integration can be transformed into line integration and
657 the interpolation can get rid of the restraints from the background mesh. Meanwhile, to handle large
658 deformation problems more conveniently, the alpha shape method is introduced to track the
659 boundary automatically, with the results shown accurate and convenient. For the purpose of
660 facilitating programming, both quasi static version and dynamic version of computational processes

661 are given. Moreover, the influence of key parameters on precision is explored systematically, and the
662 recommend values of these parameters are given in this paper. Then, two geotechnical examples,
663 slope and footing, are analyzed to further demonstrate the accuracies and reliabilities of the method.
664 Finally, by simulating the 2-D soil collapse experiment with a large run out distance, the capacity of
665 dealing with large deformations is further proved. To sum, the adaptive CTM-RPIM is a promising
666 method and has great potential in the application of geotechnical engineering.

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